

Apparatus at a draw frame having a drawing mechanism for
the doubling and drafting of fibre slivers.

CROSS REFERENCE TO RELATED APPLICATION

5 This application claims priority from German Patent
Application Nos. 102 42 391.1 and 103 29 837.1, which are
incorporated herein by reference.

BACKGROUND OF THE INVENTION

10 The invention relates to an apparatus at a draw frame or
other textile machine having a drawing mechanism for the
doubling and drafting of fibre slivers.

 Certain known forms of draw frame have a drawing
mechanism frame for accommodating the drawing mechanism,
15 which has at least two pairs of rollers each comprising an
upper roller and a lower roller, and means for adjusting
the spacing of at least one of the lower rollers in
relation to another lower roller, in each case having a
mounting device for accommodating the lower roller, and
20 lower rollers are arranged to be driven by at least one
drive element endlessly revolving around pulley wheels.

 In a known apparatus (DE-OS 20 44 996), the mountings of
the intake and middle lower rollers are displaceable on the
frame of the machine so that the extent of the drawing zone
25 can be matched to the particular fibre staple. A tensioning
pulley wheel, which is displaceable in a guideway in the

frame of the machine, allows the length of the toothed belt to be modified in accordance with the changed spacing between the axes of the middle roller and a guide pulley wheel, brought about by displacement of the intake roller.

5 The middle roller is driven by a further toothed belt. The latter toothed belt is tensioned by a tensioning pulley wheel which is fastened to the machine frame and which can pivot about one axis; as a result, it can also be matched to changed spacings between the axes of the intake roller
10 and middle roller. It is disadvantageous that displacing devices for displacement of the intake roller and the middle roller and additional tensioning devices for re-tensioning of the toothed belts after the displacement operations are necessary, requiring a considerable outlay
15 in terms of construction. In addition, it is disadvantageous that a number of work steps are required for the displacement operations and the subsequent re-tensioning operations. The belt tension is destroyed by the displacement process. Where the displacement is carried out
20 manually, spacers are inserted between the mountings, the mountings being pushed against the spacers so that, in this case too, the amount of set-up work is considerable. Finally, the displacement and re-tensioning operations result in a doubling of potential error sources when
25 setting the spacings and belt tensions.

It is an aim of the invention to provide an apparatus of the kind described at the beginning that avoids or mitigates the disadvantages mentioned and that especially is of simple construction and allows a considerable
5 reduction in the work and time required for adjustment of the lower roller(s) and, accordingly, of the extent(s) of the drawing zone(s).

SUMMARY OF THE INVENTION

10 The invention provides a drawing mechanism having a drawing mechanism frame, at least two pairs of rollers each comprising an upper roller and a lower roller and having a mounting device for accommodating the lower roller, means for adjusting the spacing of at least one of the lower
15 rollers in relation to another lower roller, and at least one drive device comprising a drive element endlessly revolving around pulley wheels, wherein the drive device can be used for adjusting the position of said at least one lower roller.

20 The measures according to the invention make it possible, by simple means, for the mountings and, as a result, the extents of the drawing zones (nip line spacings) to be adjusted in a short time. For the purpose of adjusting the extents of the drawing zones, elegant use
25 is made of existing structural elements necessarily present in the draw frame, for example, a pulley wheel and the

drive belt. Separate apparatuses for adjustment are not required. As a result of the fact that the drive belt can be in tension before, during and after adjustment, further apparatuses for re-tensioning the drive belt after the
5 adjustment are not required, which allows the extents of the drawing zones of the drawing mechanism to be changed in a short time by means that are especially simple in terms of construction.

Advantageously, a said mounting device of a said lower
10 roller is adjustable by means of a moving force applied to a pulley wheel of said drive device, which moving force is converted into an adjusting movement for the mounting device. As well or instead, a said mounting device of a said lower roller is advantageously adjustable by means of
15 a moving force applied to a drive element of said drive device, which moving force is converted into an adjusting movement for the mounting device. Advantageously, the drive element is stationary and the pulley wheel is rotated. Advantageously, the pulley wheel is stationary
20 and the drive element is moved. Advantageously, the rotation of the pulley wheel or the movement of the drive element is converted into the adjusting movement of the slider. Advantageously, at least one guide pulley wheel is attached to each slider (mounting); and the roller-driving
25 pulley wheel or guide pulley wheel(s) act, in each case one after the other, on both sides of the tensioned drive

element. Advantageously, the rotation of the pulley wheel or the movement of the drive element is accomplished manually. Advantageously, the slider is linearly displaceable.

5 Advantageously, the drive element is a toothed belt. Advantageously, an endless flexible toothed belt is present. Advantageously, the pulley wheels comprise toothed belt wheels. Advantageously, the pulley wheels comprise guide pulley wheels. Advantageously, at least one
10 driving pulley wheel is provided. Advantageously, driven pulley wheels are present. Advantageously, the drive element loops around the pulley wheels. Advantageously, the drive element and the pulley wheel are in engagement with one another. Advantageously, the pulley wheel for
15 adjustment of a slider is the drive pulley wheel of a lower roller (roller-driving pulley wheel). Advantageously, the slider is displaceable during adjustment. Advantageously, the slider is arranged to be stopped. Advantageously, the stopping arrangement is releasable. Advantageously, a
20 display device for the position of the slider is present.

Advantageously, a drive motor is used for rotation of the pulley wheel. Advantageously, a drive motor is used for movement of the drive element. Advantageously, the drive motor is used for the lower rollers. Advantageously,
25 a separate drive motor is used. Advantageously, belt shortening or belt lengthening is arranged to be

automatically evened out during adjustment.

Advantageously, the evening-out of belt length is carried out at a slider by two guide pulley wheels.

Advantageously, the lower rollers are arranged to be
5 adjusted singly and independently of one another.

Preferably, a roller-driving pulley wheel and a guide pulley wheel are attached to the slider of the intake roller and a roller-driving pulley wheel and a guide pulley wheel are attached to the slider of the middle roller.

10 Advantageously, the drive element runs around the pulley wheels at the slider of the intake roller and around the pulley wheels at the slider of the middle roller in a mirror-reflected arrangement. Advantageously, the drive element is in tension before, during and after the
15 displacement. Advantageously, the drive motor is in communication with an electronic control and regulation device. Advantageously, a measuring element is connected to the control and regulation device. Advantageously, the measuring element is capable of registering fibre-related
20 and/or machinery-related measurement variables.

Advantageously, adjustment of the slider is carried out when the drawing mechanism is in operation.

Advantageously, adjustment of the slider is carried out when the drawing mechanism is not in operation.

25 Advantageously, adjustment of the slider is carried out during can-changing. Advantageously, the draw frame is

self-adjusting. Advantageously, adjustment of the slider is carried out by inputting adjustment variables.

Advantageously, the adjustment variables can be input manually. Advantageously, a memory for adjustment

5 variables is connected to the control and regulation device. Advantageously, the slider for the intake roller and the slider for the middle roller are arranged to be connected by a rigid connecting element. Advantageously, the connecting element is releasably connected. The
10 spacing of the pairs of rollers in relation to one another may be adjustable without fibre material. The spacing of the pairs of rollers in relation to one another may be adjustable with fibre material. Advantageously, the extent of the preliminary draft zone can be adjusted.

15 Advantageously, the extent of the main draft zone can be adjusted. Advantageously, the extent of the preliminary draft zone and the extent of the main draft zone can be adjusted. Advantageously, each lower roller has its own associated drive motor. Advantageously, the intake and
20 middle lower rollers are arranged to be driven by one drive motor. Advantageously, a brake, stopping arrangement or the like is associated with the stationary pulley wheel. The brake, stopping arrangement or the like may be mechanical, electrical or electromagnetic. Advantageously,
25 the drive motor is a self-braking motor. Advantageously,

the drive motor drives a further drive train, which has a free-wheel arrangement or the like.

Advantageously, the mounting device consists of the mounting and the slider. The mounting and the slider may
5 be fastened to one another, for example by bolts. The mounting and the slider may be of integral construction.

The invention further provides an apparatus at a draw frame having a drawing mechanism for the doubling and drafting of fibre slivers, having a drawing mechanism frame
10 for accommodating the drawing mechanism, which has at least two pairs of rollers each comprising an upper and a lower roller, having means for adjusting the spacing of at least one of the lower rollers in relation to another lower roller, in each case having a mounting device for
15 accommodating the lower roller, wherein lower rollers are arranged to be driven by at least one drive element endlessly revolving around pulley wheels, characterised in that at least one pulley wheel and the tensioned drive element are used for adjusting the mounting device, wherein
20 a moving force applied to the pulley wheel or to the drive element can be converted into the adjusting movement for the mounting device.

The invention further provides a draw frame comprising a drawing mechanism according to the invention.

BRIEF DESCRIPTION OF THE DRAWINGS

5 Fig. 1 is a diagrammatic side view of an
autoleveller draw frame for use with an
apparatus according to the invention
together with a general circuit
diagram;

10 Fig. 2 is a perspective view of a side of the
draw frame showing the displaceable
mounting of the intake and middle lower
rollers;

 Figs. 3a and 3b show the drive for the intake and
15 middle lower rollers for the draw frame
according to Fig. 1, in a side view
(Fig. 3a) and plan view (Fig. 3b);

 Fig. 3c is a partial side view of a drive belt;

 Figs. 4a to 4d show, in diagrammatic form, the
20 sequential procedure for shortening of
the preliminary and main draft zones;

 Figs. 5a and 5b show the intake and middle lower
rollers before displacement (Fig. 5a)
and after displacement (Fig. 5b);

Figs. 6a and 6b show, in diagrammatic form, an electromagnetic braking apparatus for a toothed belt wheel;

Fig. 7 shows a locking device for a slider;

5 Fig. 8 shows a connection element (bridge) for connecting two sliders;

Fig. 9 is a partial side view of an embodiment comprising a drawing mechanism having three roller combinations, each having its own drive motor;

10 Fig. 10 is a side view of a drawing mechanism with input devices for manual and/or memory-assisted input of adjustment values for changing the nip line spacings in the drawing mechanism; and

15 Fig. 11 is a front view of a roller pair with an upper roller lifted off from a lower roller.

20 DESCRIPTION OF THE PREFERRED EMBODIMENTS

In accordance with Figure 1, a draw frame 1, for example a draw frame known as an HSR draw frame (trade mark) made by Trützschler GmbH & Co. KG, has a drawing
25 mechanism 2, upstream of which is an intake 3 of the drawing mechanism and downstream of which is an exit 4 from

the drawing mechanism. The fibre slivers 5, coming from cans (not shown), enter the sliver guide 6 and, drawn by the draw-off rollers 7, 8, are transported past the measuring element 9. The drawing mechanism 2 is designed as
5 a 4-over-3 drawing mechanism, that is to say it consists of three lower rollers I, II, III (I delivery lower roller, II middle lower roller, III intake lower roller) and four upper rollers 11, 12, 13, 14. Drafting of the fibre sliver combination 5' from a plurality of fibre slivers 5 is
10 carried out in the drawing mechanism 2. Drafting is composed of preliminary drafting and main drafting. The roller pairs 14/III and 13/II form the preliminary draft zone and the roller pairs 13/II and 11, 12/I form the main draft zone.

15 The attenuated fibre slivers 5 reach a web guide 10 in the exit 4 from the drawing mechanism and, by means of the draw-off rollers 15, 16, are drawn through a sliver funnel 17, in which they are combined to form one fibre sliver 18, which is then deposited in cans. Reference
20 letter A denotes the work direction.

The draw-off rollers 7, 8, the intake lower roller III and the middle lower roller II, which are connected to one another mechanically, for example by toothed belts, are driven by the control motor 19, it being possible, in the
25 process, for a desired value to be specified. (The associated upper rollers 14 and 13, respectively, revolve

by virtue of the motion of the lower rollers.) The delivery lower roller I and the draw-off rollers 15, 16 are driven by the main motor 20. The control motor 19 and the main motor 20 each have their own controller 21 and 22, respectively. Control (speed-of-rotation control) is carried out in each case by means of a closed control loop, a tachogenerator 23 being associated with the control motor 19 and a tachogenerator 24 being associated with the main motor 20. At the intake 3 of the drawing mechanism, a variable proportional to the weight of the fibre slivers 5 fed in, for example their cross-section, is measured by an intake measuring element 9 known, for example, from DE-A- 44 04 326. At the exit 4 from the drawing mechanism, the cross-section of the delivered fibre sliver 18 is ascertained by an exit measuring element 25 associated with the sliver funnel 17 and known, for example, from DE-A- 195 37 983. A central computer unit 26 (control and regulation device), for example a microcomputer with a microprocessor, sends a setting for the desired value for the control motor 19 to the controller 21. The measurement values of the two measuring elements 9 and 25 are sent to the central computer unit 26 during the drawing process. The desired value for the control motor 19 is determined in the central computer unit 26 from the measurement values of the intake measuring element 9 and from the desired value for the cross-section of the delivered fibre sliver 18. The

measurement values of the exit measuring element 25 are used for monitoring of the delivered fibre sliver 18 (delivered sliver monitoring). By means of this control system, it is possible for variations in the cross-section of the fibre slivers 5 fed in to be compensated, and for the fibre sliver to be made more uniform, by appropriately regulating the drafting process. Reference numeral 27 denotes a display monitor, 28 an interface, 29 an input device, 30 a pressure rod and 31 a memory.

With reference to Fig. 2, each of lower rollers II, III has an associated mounting device comprising a respective mounting 33a, 34a. The trunnions Ia, IIa, IIIa (see Fig. 3b) of the lower rollers I, II and III are mounted so as to be capable of rotation in mountings 32a, 33a, 34a (32b, 33b, 34b are located on the other side of the drawing mechanism and are not shown). The mountings 33a and 34a are bolted onto sliders 35a and 36a, respectively, which are displaceable in the direction of the arrows C, D and E, F, respectively, along a bar 37a. The two ends of the bar 37a are fixedly mounted in mounting blocks 38' (38'' not shown), which are attached to the frame 39 of the machine.

Displacement of the sliders 35a, 35b; 36a, 36b at the same time causes the mountings 33a, 33b; 34a, 34b and, as a result, the lower rollers II and III, respectively, to be displaced and moved in directions C, D and E, F, respectively. The associated upper rollers 13 and 14 are

correspondingly moved (in a manner not shown) in directions C, D and E, F, respectively. By that means, the nip line spacings between the roller combinations are modified and set.

5 Locking of the sliders 35a, 35b; 36a, 36b is accomplished by means of a catch device, stopping device or the like, one suitable form of stopping device being shown in Fig. 7.

Referring to Fig. 3a, the lower rollers II and III are
10 driven from the right-hand side of the draw frame, seen in the direction of material flow A, by means of a common loop mechanism in the form of toothed belt wheels 40, 41 and a toothed belt 47. The different speeds of rotation of the lower rollers II and III are achieved by means of change-
15 gearwheels at the drive trunnions Iia, IIIa provided with different numbers of teeth. The toothed belt 47 runs in direction B (that is to say contrary to the work direction) onto the control drive, which is in the form of a servo motor 19. The lower roller I is driven from the left-hand
20 side of the machine by means of a loop mechanism in the form of toothed belt wheels and a toothed belt 47'. For that purpose, the toothed belt 47' runs on the left-hand side from the toothed belt disc 40' at the lower roller I in direction G onto the servo motor 20.

25 In operation, that is to say when the fibre slivers are running in direction A, the toothed belt 47 moves in

direction G. Starting from the toothed belt wheel 47 arranged on the drive motor 19, the toothed belt 47 runs successively over a toothed belt wheel 45, a smooth guide pulley wheel 46, the toothed belt wheel 40 (roller-driving pulley wheel for the lower roller III), the toothed belt wheel 41 (roller-driving pulley wheel for the lower roller II), a smooth guide pulley wheel 42 and a toothed belt wheel 43. As shown in Fig. 3c, the belt 47 has a toothed side 47a and a smooth side 47b. By means of its teeth, the toothed belt 47 is in positive engagement with the toothed belt wheels 40, 41, 43, 44 and 45. The smooth side (reverse) of the toothed belt 47, opposite the toothed side, is in contact and in engagement with the smooth guide pulley wheels 46 and 42. The toothed belt 47 loops around all the pulley wheels 40 to 46. In operation (when the fibre slivers are running in direction A during drafting), the toothed belt wheels 40, 41, 43, 44 and 45 rotate clockwise and the guide pulley wheels 42 and 46 rotate anti-clockwise.

The toothed belt wheels 40, 41 are associated with the mountings 34a and 33a, respectively, whereas the guide pulley wheels 42, 46 are attached to the sliders 35a and 36a, respectively, in a manner allowing rotation. Because of the rigid attachment between the mounting 34a and the slider 36a and between the mounting 33a and the slider 35a (for example, by means of bolts), there are associated with

the lower rollers II and III, in each case, one toothed belt wheel 40 and 41, respectively, and one guide pulley wheel 46 and 42, respectively. The toothed belt 47 runs around the pulley wheels 40, 46, on the one hand, and
5 around the pulley wheels 41, 42, on the other hand, in a mirror-reflection arrangement (see Fig. 3b).

The zone between the pairs of rollers 13/II and 14/III is designated VV (preliminary drafting) and the zone between the pairs of rollers 12/I and 13/II is designated
10 HV (main drafting) (see Fig. 4a). When, in accordance with Fig. 3a, the nip line spacing between the roller pairs 14/III and 13/II is to be increased, at least one pair of rollers must be moved away from the respective other pair of rollers. For that purpose the slider 35a may
15 be displaced towards the right, which may be accomplished in two ways:

a) The slider 35a is unlocked. A pulley wheel, for example the toothed belt wheel 44, is stopped so that there is no possibility of rotation. Stopping
20 may be accomplished, for example, by mechanical or electromagnetic means. As a result the toothed belt 47 is stationary and cannot be moved. The toothed belt wheel 41 is then rotated anti-clockwise, for example manually using a crank or
25 the like, whereupon the guide pulley wheel 42 likewise rotates, clockwise, as a matter of

necessity. In the process, the rotary movement of the toothed belt wheel 41 is converted into a longitudinal movement of the slider 35a in direction C, the toothed belt wheel 41 and the guide pulley wheel 42 winding along opposite sides of the stationary toothed belt 47, thereby "shortening", as it were, the toothed belt 47 at one pulley wheel and "lengthening" it at the other pulley wheel. The length of belt required during that "winding along" at the toothed belt wheel 41 is made available at the guide pulley wheel 42. The lower roller II is thereby displaced in direction C by means of the slider 35a and the mounting 33a.

b) The slider 35a is unlocked. The toothed belt wheel 41 is stopped so that there is no possibility of rotation. As a result the guide pulley wheel 42 is also stopped of necessity. Then, clockwise rotation is brought about by means of the drive motor 19. The toothed belt 47 moves in direction G, likewise "shortening" the belt 47 at one pulley wheel and "lengthening" it at the other pulley wheel. The length of belt actually required between the toothed belt wheels 40 and 41 is made available between the toothed belt wheels 43 and pulley wheel 42. The rotary movement

of the toothed belt wheel 44 and the movement of the toothed belt 47 is thereby converted into a longitudinal movement of the slider 35a in direction C. The lower roller II, mounted in the mounting 33a (which is rigidly connected to the slider 35a), is likewise moved in direction C as a result.

In practice, it is often the case that, in accordance with Figs. 4a to 4d, first the preliminary draft zone VV is modified and then the main draft zone HV. In the case of shortening of the draft zones VV and HV, the slider 36a is displaced in the direction of the arrow E from the position according to Fig. 4a into the position according to Fig. 4b. As a result, the nip line spacing in the preliminary draft zone VV is reduced from "a" to "a'". Then, in accordance with Fig. 4c, the sliders 36a and 35a are rigidly connected to one another by means of a bridge 50. Finally, the rigidly coupled sliders 36a and 35a are moved, in accordance with Fig. 4d, in the direction of the arrows E and C, from the position shown in Fig. 4c into the position shown in Fig. 4d. As a result, the nip line spacing in the main draft zone HV is shortened from "b" to "b'". - A corresponding procedure is used in the case of lengthening the preliminary and main draft zones, that is to say the coupled sliders 35a and 36a are displaced in the direction of the arrows F and D (see Fig. 2), as a result

of which the main draft zone HV is lengthened. Then, the sliders 35a and 36a are uncoupled from the bridge 50. Finally, the slider 36a is moved in the direction of the arrow F (see Fig. 2), as a result of which the preliminary
5 draft zone VV is lengthened.

With regard to the fibre slivers 5 in the drawing mechanism 2, it should be noted that, in the case of shortening of the draft zones VV and HV, a small amount of stretching, in direction B, of the fibre slivers 5^{IV}
10 upstream of the pair of rollers 14/III can occur on displacement in accordance with Figs. 4a, 4b, but because of the length (about 1.5 m) of the spacing between the transport rollers 7, 8 and the pair of rollers 14/III this is without significance. In the case of shortening, a
15 sagging loop does not form in the preliminary draft zone VV because in the case of displacement referring to the pairs of rollers 14/III and 13/II either one or both pairs of rollers are rotatable because the drives to both pairs of rollers are coupled by way of the toothed belt 47. In
20 contrast, in the case of shortening of the main draft zone HV, a sagging loop is formed in fibre slivers 5'', which is drawn out or drawn straight by rotation of the pair of rollers 12/I in the work direction A by means of the main motor 20. - In the case of lengthening of the draft zones
25 VV and HV, the pair of rollers 12/I is, in a first step, rotated backwards in direction B, whereupon a sagging loop

is intentionally formed in the fibre slivers 5''. When the main draft zone HV is subsequently lengthened by displacement of the coupled sliders 35a and 36a in direction D and F, the artificially formed loop is, in the process, once again drawn out or drawn straight. Finally, after uncoupling of the bridge 50, the slider 36a is displaced in direction F. As a result of the above-mentioned coupling of the drives to the intake and middle lower roller pairs by means of the toothed belt 47, the length of the fibre slivers 5' in the preliminary draft zone VV remains unaffected. Possible slight longitudinal compression of the fibre slivers 5^{IV} upstream of the pair of rollers 14/III is, in respect of the drafting and the constitution of the fibre slivers 5^{IV}, without significance.

Figs. 5a, 5b show a suitable construction for bringing about the displacement of the sliders 36a and 35a. The nip line spacing in the preliminary draft zone VV is lengthened from "a" (Fig. 5a) to "a'" (Fig. 5b). The sliders 36a and 35a are displaced one after the other according to the arrows E and C, respectively. Displacement is accomplished by stopping the toothed belt wheel 40 or fixing it with a holding brake or the like and then actuating the drive motor 19, whereupon the toothed belt 47 moves. In continuation thereof, the sliders 36a and 35a are displaced

in accordance with Figs. 4a, 4b and, subsequently,
Figs. 4c, 4d.

In Fig. 6a there is shown an electromagnetic holding
brake for braking the toothed belt wheel 44. The brake has
5 a rod-shaped iron core 53 surrounded by a plunger coil 54.
Mounted on one end face of the iron core 53 is a brake
shoe 55, for example made of plastics material or the like.
The iron core 53 is displaceable in the direction of the
arrows M, N. When current flows through the plunger
10 coil 54, the iron core 53 is moved in direction M, in
accordance with Fig. 6b, so that the brake shoe 55 is
pressed against the smooth cylindrical surface of the
shaft 44a of the toothed belt wheel 44. As a result, the
toothed belt wheel 44 is fixed (stopped) so that it cannot
15 rotate, for as long as voltage is applied to the plunger
coil 54.

In Fig. 7 there is shown a stopping device for slider
36a and corresponding lower roller III. A pneumatic
cylinder 60 having a piston rod 61 is attached to the
20 slider 36a. When subjected to pressure from the pneumatic
cylinder 60, the piston rod 61 is moved out in the
direction of the arrow P and comes to rest, with a high
degree of contact pressure, against the machine frame 61.
The slider 36a is fixed (stopped) so that it cannot be
25 displaced with respect to the bar 37a, for as long as
compressed air is applied to the pneumatic cylinder 60.

Lower roller II may be provided with an analogous arrangement.

In accordance with Fig. 8, there is provided, as the bridge 50 between the sliders 35a and 36a, a flat piece of metal (plate), which is fastened in the region of one of its ends 50a to the slider 36a, for example using bolts. In its region 50b facing the slider 35a, the flat piece of metal has an elongate hole 50c, through which a bolt 62 can engage in a threaded hole (not shown) in the slider 35a. By means of this bridge 50, the sliders 35a and 36a can be rigidly connected to one another, releasably, at different spacings with respect to one another.

In accordance with Fig. 9, in contrast to Fig. 1, each lower roller I, II and III is driven by its own drive motor 20, 52 and 19, respectively, as shown, for example, in DE-OS 38 01 880. The motor 20 drives the toothed belt wheel 55 of the lower roller I by way of the toothed belt 56; the motor 52 drives the toothed belt wheel 41 of the lower roller II by way of the toothed belt 57; and the motor 19 drives the toothed belt wheel 40 of the lower roller III by way of the toothed belt 47. Attached to the slider 36a, in addition to the smooth guide pulley wheel 46, is a further smooth guide pulley wheel 51. The endless toothed belt 47 loops around, in succession, the pulley wheels 44, 46, 40, 51 and 43. The toothed belt wheels 44, 40 and 43 are in engagement with the teeth of

the toothed belt 47, whereas the smooth guide pulley wheels 46 and 51 are in engagement with the smooth reverse side of the toothed belt 47. The sliders 35a and 36a are rigidly connected to one another, releasably, by means of
5 the bridge 50. When they are not connected by the bridge 50, the sliders 35a and 36a are individually displaceable and when they are connected by the bridge 50 they are jointly displaceable.

In accordance with Fig. 10, the drive motor 19 for
10 lower rollers II and III is in communication with the electronic control and regulation device 26. Adjustment values for modification of the draft zones VV and HV (that is to say the extents of the drawing zones) either can be entered manually by way of the input device 29 or can be
15 called up from a memory 31 for particular categories of fibre material.

Adjustment of the nip line spacing in the preliminary draft zone VV and/or the main draft zone HV can be carried out with the fibre slivers 5 inserted.

20 Displacement can be carried out with the upper rollers 11 to 14 in the loaded state. Figs. 1 and 10 show inserted fibre slivers 5 and loaded upper rollers 11 to 14. With the fibre slivers inserted and the upper rollers 11 to 14 loaded, the sliders 35a, 36a or mountings of at least one
25 lower roller II, III are unlocked, the sliders or mountings are set to the desired nip line spacing a , a' ; b , b' by

means of a displacement device, for example in accordance with Figs. 3a, 3b; 5a, 5b and then the sliders 35a, 36a or mountings are locked again (for example in accordance with Fig. 7).

5 Displacement can also be carried out with the upper rollers 11 to 14 lifted off. The upper rollers 11 to 14 may be lifted off completely from the lower rollers I to III in the manner shown in DE-OS 197 04 815, the upper roller 14 being swung out on a portal 58 about a pivot mounting 59.

10 However, it may also be sufficient for the upper rollers 11 to 14 to be unloaded and to be lifted off from the lower rollers I to III only to a slight degree such that the fibre slivers 5 are not caught by the pairs of rollers during displacement of the draft zones VV and HV but can

15 slide through the roller nip without being adversely affected.

The invention has been illustrated using the example of the adjustment of the nip line spacings of a drawing mechanism of a draw frame. It likewise encompasses the adjustment of

20 drawing mechanisms of other machines, for example carding machines, combing machines, fly frames and ring spinning frames.